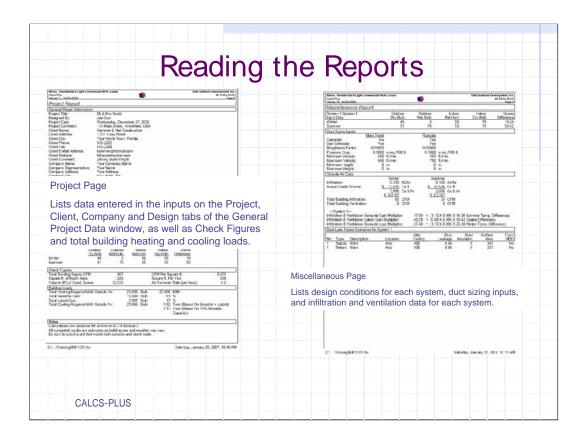
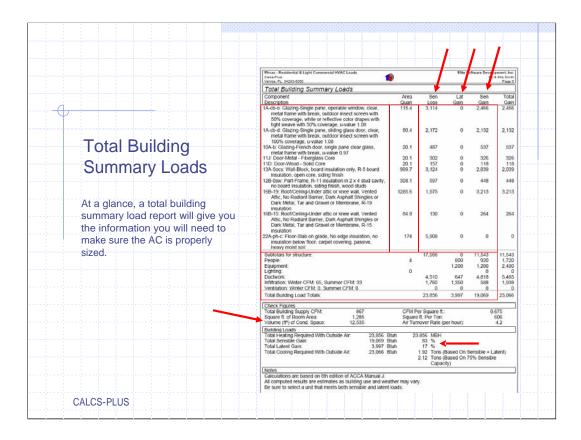
Rig	ght Sizing AC Systems for
<b>+</b>	Profit and Energy Star
	Certification
	Part II
	RESNET 2007
	Dennis J Stroer
	CALCS-PLUS
	Venice Florida
CALCS-PL	US



The calculation is finished (for now) so it's time to make some sense out of the reports. The practitioner who understands the results of the calculation will use it as a tool for consulting and to identify problems having to do with system sizing, airflow, temperature un-balances, etc.

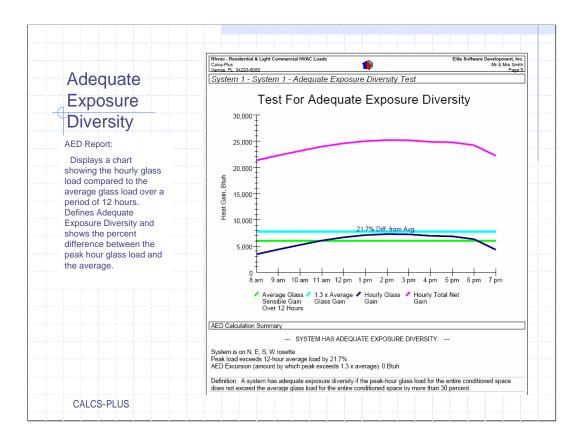
All calculation programs (ACCA certified that is) gives general or miscellaneous information that pertains to the project location, who it is for, who did the calculation, some check figures, etc. But they don't mean much unless you have the rest of the meat of the calculation to back it up.



At a glance, a total building summary load report will give you the information you will need to make sure the AC is properly sized. For heating you will look at the sensible loss and for cooling the sensible and latent gain must be considered. The report will give the materials used, area or quantity, and the loss or gain of those materials. It will also include internal gains, duct gains, and infiltration/ventilation gains that were used to calculate the total HVAC load. Other important information on this summary should include conditioned area & volume and the cooling sensible heat ratio (SHR) of the building load.

				ary fo							
7	Rhvac - Residential & Light Commercial HVAC Loads Calcs-Plus Venice, FL 34293-6060								Elite Software Development, Inc Mr & Mrs Smith Page 7		
	System 1 Room Lo	Load Summary									
	Room No Name	Area SF	Htg Sens Btuh	Htg Nom CFM	Run Duct Size	Run Duct Vel	Clg Sens Btuh	Clg Lat Btuh	Clg Nom CFM	Air Sys CFM	
	Zone 1 1 Bedroom 1 2 Bedroom 2 3 Bedroom 3	155 155 120	3,376 3,376 1,680	36 36 18	1-6 1-6 1-4	465 465 580	2,006 2,006 1,113	246 246 113	91 91 51	91 91 51	
	4 Bathroom Powder Area 5 Bathroom Tub	48	73	10	1-4	84 287	162 550	77	7 25	7 25	
	Area 6 Great Room 7 Kitchen / Dining	368 276	5,629 4,618	59 49	1-10	506 462	6,071 4,484	631 1,406	276 204	276 204	
	8 Laundry Duct Latent	120	4,114	43	1-7	456	2,678	631	122	122	
	System 1 total Cooling System Summary	1,285	23,856	251			19,069	3,997	867	867	
-~	Cooling Gystem Gummary	Cooling Tons	Sens	ible/Latent Split		Sensible Btuh		Latent Btuh		Total Btuh	
	Net Required: Recommended:	1.92 2.12		33% / 17% 75% / 25%	5 5	19,069 19,069	i i	3,997 6,356	5 5	23,066 25,426	

The room summary report should give the area, heating BTUH, heating CFM, runout duct size (if you are using that feature), run-out duct velocity (again if you are using that feature), sensible cooling BTUH, latent cooling BTUH, nominal cooling CFM, and system CFM for each room (if it was set to a fixed CFM).

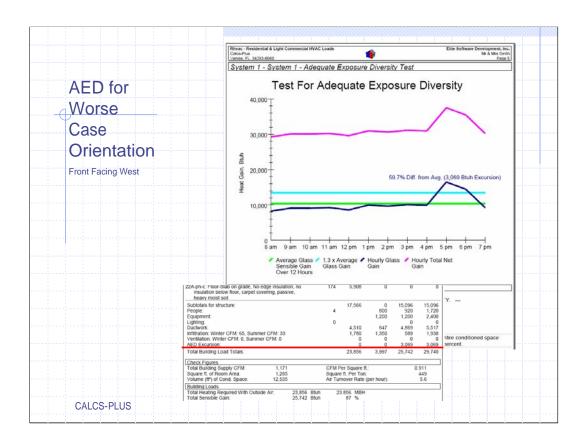


Adequate exposure diversity (AED) is the comparison of the peak load to the average load. The chart above shows the average sensible gain of the glass (the green line) over a 12 hour period during the day (8AM to 7PM). The cyan colored line marks where 30% above the average glass load would follow. The dark blue line is the actual glass load during various times of the day. Note that the in this house the glass peaks out sometime between 1 and 3 PM. Also notice that the peak load does not go above the 30% line, it falls shore by about 8.3%. The hourly glass gain curve is positioned at the start of the average load line which projects the deviation of the total building load during the day with respect to peak load.

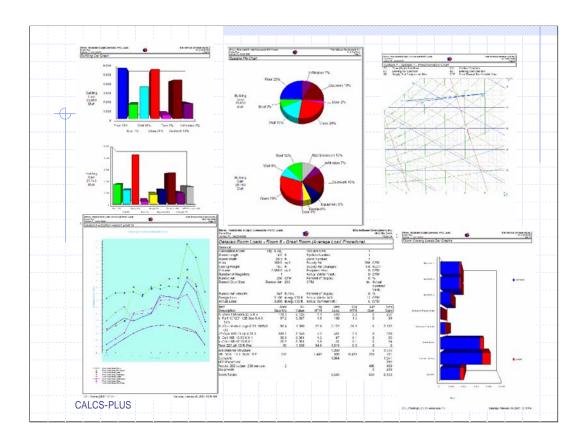
Building Rhvac - Residential & Light Commercial HVAC Loads						Elite Software Development, Inc. Mr & Mrs Smith					
Rotation	Venice, FL 34293-6060 Page 8  Building Rotation Report										
Report	All rotation degree values in this report are clockwise with respect to the project's original orientation.  Building orientation as entered (zero degrees rotation): Front door faces South										
	Individual Rooms										
	Rm. Roo		0° Rot. CFM	45° Rot. CFM	90° Rot. CFM	135° Rot. CFM	180° Rot. CFM	225° Rot. CFM	270° Rot. CFM	315° Rot. CFM	High Duct Size
	System 1 Zone 1:	:									
	1 Bed 2 Bed	room 1 room 2 room 3	91 91 51	104 134 72	*144 *144 *78	130 100 55	91 91 51	123 95 52	133 133 71	102 132 71	1-7 1-7 1-5
		room Powder	7	8	/ o *8	8	7	7	7 7	8	1-3
	Are 5 Bat	a nroom Tub Area	25	27	*36	34	25	32	34	26	1-4
	6 Gre	at Room hen / Dining	276 204 122	296 290 133	298 *315 *147	*435 236 138	276 204 122	411 223 131	274 289 135	292 286 131	1-12 1-10 1-7
	* Indicates highest CFM of all rotations.										
	Whole Bu	lding									
	Rotation	Front door Faces		Supply	Ser	nsible	Late			Recomn	
	Degrees 0°	South		867		Gain 9,069	3,9		Tons 1.92		Tons 2.12
	45° 90°	Southwest West		1,064 *1,171		3,403 5.742	3,9 3.9		2.28 *2.48		2.60
	135°	Northwest		1,136		4,981	3,9		2.41		2.78
	180°	North		867		9,069	3,9		1.92		2.12
	225° 270°	Northeast East		1,075 1,075		3,629 3,631	3,9 3,9		2.30 2.30		2.63
	315°	Southeast		1,050		3,081	*3,9		2.26		2.56
	* Indicates	highest value of	all rotations	s							

If you do consulting for a builder that has several models that will be placed in a subdivision in any one of the 8 orientations this report will be a real value. In our initial set-up we told the program that the front door would face south. The above report shows that the worse direction the home can face is west. The load on the building increases from 2.12 tons (recommended) when south facing to 2.86 tons (recommended) facing west. Lets see why.

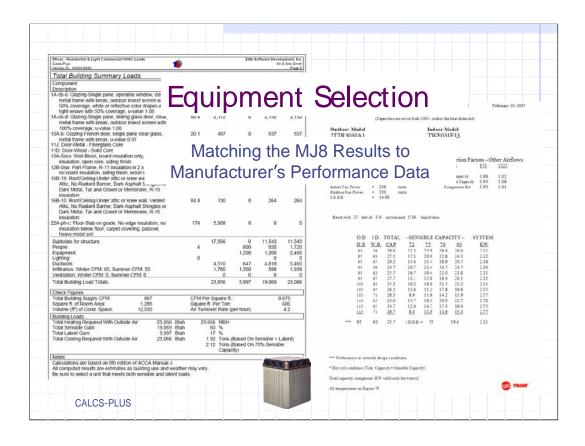
Note: Recommended Tons will be discussed in the section on Equipment selection.

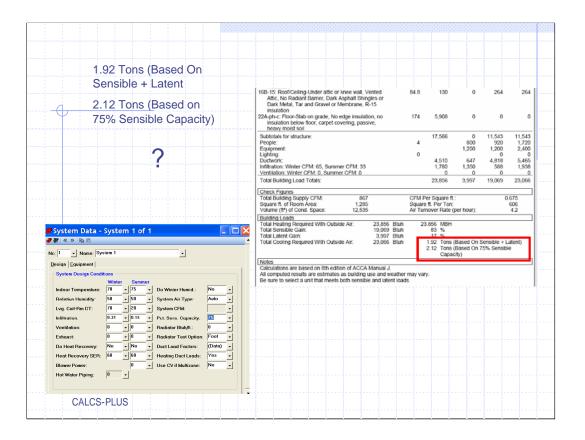


Here is the AED test for the home after we have rotated the front from south to west. The glass peaks over the 30% line by 3,069 BTUH. When this happens we call the 3,069 the "Excursion and it is added back to the calculation. This can be seen on the output with the total building summary loads. If building does not have AED the Excursion becomes a line item in the "Total Building Load Totals". If the building does have AED the Excursion line is not added to the report.



There are many other useful reports that can come out of the MJ8 calculation such as graphs that show where the loads are coming from, graphs that show room loads, Psychrometric process relating to infiltration and or ventilation, and detailed report for each room. These reports are useful when analyzing to identify problems.





This program gives a recommended tonnage size for cooling based on the idea that typical air conditioning systems remove heat at 75% sensible relative to the total BTUH output. This would also mean that the latent removal is 25% of the output. Keep in mind that a 75% SHR may not fit the model line that the owner/builder may want.

If you know the actual SHR of the equipment line that will be installed it can be entered in a "System Data" and the recommended tonnage will be based on it.

Total Building Load Totals: 23,856 3,997	TRANE RS PERFORMANCE DATA COOLING  - U.S. (ENGLESS) -  (Capacities are set in bmb/1000 - indoor fan best dedúcted)
Check Figures         667         CFM Per Square ft.           Colare ft. of Room Area:         1,285         Square ft. Per Ton:           Volume (ft*) of Cond. Space:         12,535         Air Turnover Rate (per hour):	Outdoor Model         Indoor Model           2TTR3030A1         TWE031E13           Airflow = 1000
Subding Loads	Values At ARI Rating Conditions         Correction Factors - Other Airflows           Total Net Capacity         − 27800         Bub           Airflow         − 1020         CPM           Compressor Power         − 1970         wath           Index Fax Power         − 238         wath           Congressor Kw         0.99         1.01           Outdoe Fax Power         − 150         wath           Compressor Kw         0.99         1.01
All computed results are estimates as building use and weather may vary, be sure to select a unit that meets both sensible and latent loads.	SIER = 14.00  Rated with 25 feet of 3/4 section and 5/16 liquid lines.
Manufacturers performance cooling data (like the one at the right) will give system performance at conditions other than ARI	O.D. 1.D. TOTALSENSIBLE CAPACITY SYSTEM  D.B. W.B. CAP 72 75 78 80 KW  13 39 260 213 239 260 260 211  15 65 271 173 200 22.6 243 212  15 67 292 15.4 15.4 18.9 20.7 21.6  19 39 24.7 20.7 23.4 24.7 24.7 23.0
For the area of the country this home is going to be located we will be interested in how the system will perform at or near MJ8 design conditions of 91°F outdoor and 75°F @ 50%	91 61 257 167 194 220 23.8 231 92 61 277 13.1 15.8 18.4 20.1 235 105 63 24.3 16.2 18.8 21.4 23.2 251 105 67 262 12.6 15.2 17.8 19.6 255 105 71 28.3 8.9 11.6 14.2 15.9 25.7 115 65 23.0 15.7 18.5 20.9 22.7 270 115 67 24.7 12.0 14.7 17.3 19.0 2.75
. RH.	111 71 267 84 110 136 154 2.77 *** 95 63 25.7 IDDB- 75 19.4 2.31
	*** Performance at selected design conditions  * Day cost condition (Total Capacity = Sensible Capacity)  Total Capacity, compressor KW valid only for websall  All unperstances in Degree 'F
	An amperence in pagine 2

Manufacturers' performance date shows equipment output at various outdoor and indoor conditions; conditions outside ARI testing. The difference is plainly shown on this manufacturers performance sheet for a nominal 2.5-ton system. The "Values At ARI Rating Conditions" shows that this system has a total net capacity 27,800 BTUH and a SEER rating of 14.00 at 1020 CFM. This is good information but we really need to know how the system is going to perform at design conditions. At 95°F DB outdoor, and 75°F DB & 63°F WB indoor. This system match produces 25,700 BTUH total and 19,400 BTUH sensible output which gives it a 75.5% SHR. So it will work well for this application.

	Where did 63	3°F Indoor W	/B Come Fro	om?
<b>+</b>				
4	Psychrometrics - Mixed	Air		
	Description: Mixed air conditions Elevation ft 0 Barr		29.921	
	Psychrometric Properties	Source 1 Source	ce 2 Mixed Air	
	Air Flow Rate ft³/min	1000 4	3 1043	7
	Dry Bulb Temperature F	75 7 9	75.644	j
	Wet Bulb Temperature F	62.547	63.273	]
	Relative Humidity (%):	<b>▼</b> 50	6.454 50.69	]
	Vapor Pressure psia		40704 0.22272	
	Dew Point Temperature F		3.37 56.048	
	Moisture Content: Grains/lb  Specific Volume ft*/lb	64.65		4
	Enthalpy Btu/lb		1.358 13.697 28.633	4
	Ениару Биль	28.108      4	20.833	
			<u>Calculate</u> Close	
CALCS-PL	LUS			

MJ8 indoor design conditions of 75°F DB @ 50% RH yields an indoor wet bulb temperature of 62.5. We guessed at the building infiltration rate, .16, semi tight for buildings under1500 SQ FT. This equates out to 33 CFM which doesn't meet ASHRAE 62.2 minimum of 7.5 CFM per person and .01 CFM per SQ FT. The home has three bedrooms so four people are considered which gives 30 CFM + .01 CFM x 1285 SQ FT = 43 CFM. The condition of the mixed air at the return plenum just before the evaporator coil is 75.6° DB and 63.3° WB.

## Wrap Up



- Understand the load calculation program you are using.
- Do a good take off from plans or as-built measurements.
- Understand the outputs and have faith in your work.
- Ask for manufactures performance data.
- Typically, if you choose a piece of equipment that will meet the sensible and net load the latent load will be handled by runtime.
- Do not exceed 15% of the total building cooling load.
- Remember that the cooling systems ability to control indoor relative humidity is through long run times.

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