



Calculation of AC System Performance *Without Gauges*

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testo worldwide

- Largest manufacturer of portable instrumentation in the world
- 1350 employees worldwide, 110 R&D Engineers
- testo AG - Headquartered in Black Forest Germany
- testo inc. - Headquartered in Flanders, NJ
- 25 subsidiaries worldwide (USA, Japan, Australia, China, India, Russia, Europe, ...)
- ISO 9001 Quality Standards
- Service and repair facilities worldwide
- 50 years of experience



Gauges

- Gauges = Refrigerant gauges
- Used for charging
- Omnipresent
- Only a part of the answer
 - Superheat or subcooling
 - airflow
- Limited accuracy



System Performance

- Performance (BTU/hr)
 - ...as designed by the manufacturer
 - ...as specified by the dealer/installer
 - ...as purchased by the customer
- In Heating mode
 - Heating output in BTUh or KW
- In cooling mode
 - Cooling in tons
 - BTUh or KW
 - 1ton =12,000 BTU/hr



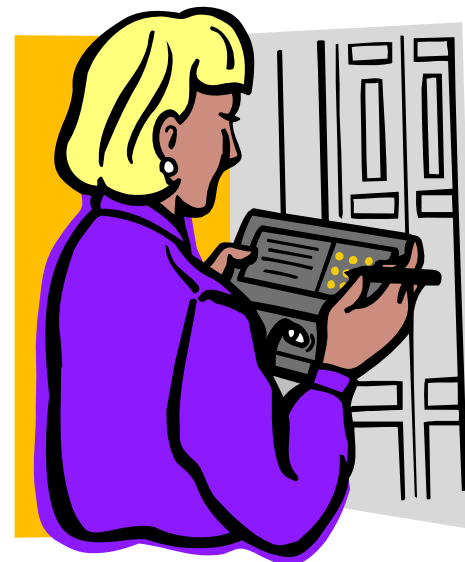
How do you do that?

- Energy conversion rate at the box
- Mass flow rate
 - Airflow over the coil
- The change in latent heat
 - Change in enthalpy



Why measure?

- Verify **appliance** operation
 - Heating capacity
 - Cooling capacity
- Verify **system** operation
 - Delivery (ducts)
 - Throw (mixing-comfort)



System Performance

- Performance cannot be assumed!!!
 - Performance varies with load conditions
 - Equipment performance does not assure delivered performance
 - Systems are field installed and require a field commissioning procedure
- Efficiency and performance go hand in hand

Capacity & Efficiency

- Capacity and efficiency are directly related
- With low efficiency you have low capacity
- Verification of rated capacity guarantees performance.
- A quick determination can be made where the problem lies.
 - The equipment
 - The duct system
 - Or both!



Benefits

- AC

- Better humidity removal and comfort resulting in higher thermostat settings
- Increased cooling capacity resulting in shorter run times
- Equipment will cycle off during peak demand periods
- Reduced electrical energy consumption

- Heating

- Reduced short cycling
- Extend equipment life
- Reduced fuel usage



Who should measure

- Contractors
 - Troubleshoot customer complaints
 - Proper installation and set up
 - System commissioning
- Energy/program auditors
 - Test-out
 - Verify performance
 - Virtual VSP



What happens today

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LEFT BLANK*



Estimating vs. measuring

- Air flow estimates
 - Pressure drop across the dry evaporator coil
 - Total external static pressure method
 - The temperature rise method (Sensible heat formula)
 - RPM and manufacturers' fan curves (Belt or VF Drive)
- Airflow measurements
 - Pitot tube and digital manometer
 - Hot Wire Anemometer
 - Mini-vane Anemometer

Limitations today

- Estimations
- Surrogate indicators
- Understanding measurements are possible and helpful
- The Holy Grail
 - An accurate and reliable way to measure **airflow** in the field
 - That doesn't cost an arm and a leg
 - And will be used more often than not



The Sensible Heat Formula

$$\text{BTU}_h = \text{SPECIFIC HEAT} \times \\ \text{SPECIFIC DENSITY} \times \\ 60 \text{ MIN/HR} \times \text{CFM} \times \Delta T$$

Or

$$= 0.24 \times 0.75 \times 60 \times \text{CFM} \times \Delta T$$



$$= 1.08 \times \text{CFM} \times \Delta T$$



Total Heat Formula

Heating or Cooling

$$\text{BTU/hr} = \text{specific density} \times 60 \text{ min/hr} \times \Delta h$$
$$= 0.075 \times 60 \times \text{CFM} \times \Delta h$$



$$= 4.5 \times \text{CFM} \times \Delta h$$



Standard Air

- 0.075 lbs./ft³ at
 - 68 degrees F
 - 0% rH
 - 14.7 psia

But.....

Air is never standard!!!

- The constants 1.08 & 4.5 are derived from standard air density at 0.075 pounds/ft³
- If we want accurate calculations, we must correct our measurements for **actual** air density!!!
- Pitot tube and hot wire anemometers are density dependent



Off-line calculation of heating & cooling capacity

Don't Panic

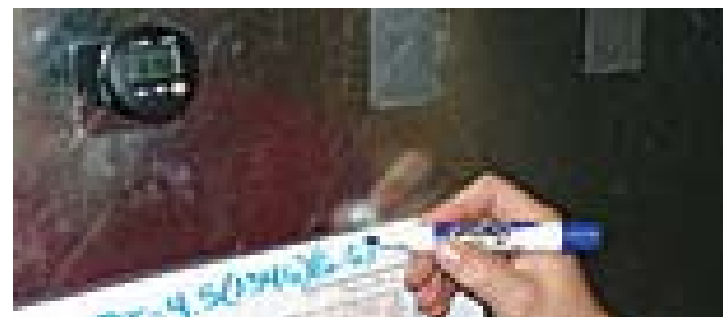
It's as easy as

- 1 Measure**
- 2 Look up**
- 3 Calculate**



Off-line: STEP 1: Measure

- Airflow - CFM
 - in the return
- Entering and leaving wet bulb
 - Around the heat exchanger or A-coil
 - Allows you to get the enthalpy change (total energy change)

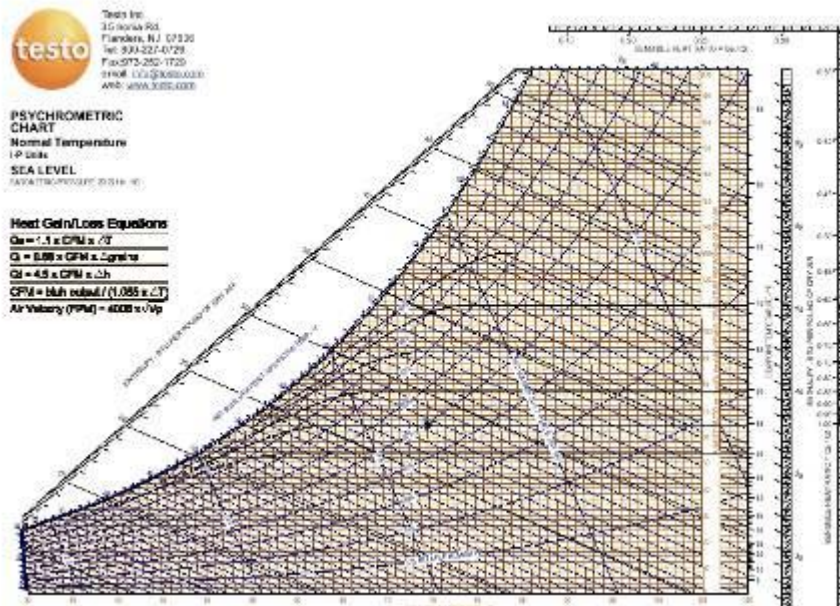




Off-line: STEP 2: Look up

- Find change in enthalpy (Δh) from wet bulb measurements
- Using a table, chart or program

WET BULB TEMPERATURE	ENTHALPY CHART WET BULB TO ENTHALPY CONVERSION						
	Wet bulb temperature in tenths of a degree Fahrenheit						
	0	0.1	0.2	0.3	0.4	0.5	0.6
35	12.97	13.01	13.05	13.10	13.14	13.18	13.23
36	13.40	13.45	13.49	13.53	13.58	13.62	13.67
37	13.85	13.89	13.93	13.98	14.02	14.07	14.11
38	14.30	14.34	14.39	14.43	14.48	14.52	14.57
39	14.75	14.80	14.84	14.89	14.94	14.98	15.03
40	15.21	15.26	15.31	15.36	15.40	15.45	15.50
41	15.69	15.73	15.78	15.83	15.88	15.92	15.97
42	16.16	16.21	16.26	16.31	16.36	16.41	16.45
43	16.65	16.70	16.75	16.80	16.85	16.90	16.95
44	17.14	17.19	17.24	17.29	17.34	17.39	17.45
45	17.65	17.70	17.75	17.80	17.85	17.90	17.95
46	18.16	18.21	18.26	18.31	18.37	18.42	18.47
47	18.69	18.73	18.79	18.84	18.89	18.94	19.00
48	19.21	19.26	19.32	19.37	19.43	19.48	19.53
49	19.75	19.81	19.86	19.92	19.97	20.03	20.08
50	20.30	20.36	20.41	20.47	20.53	20.58	20.64
51	20.86	20.92	20.98	21.04	21.09	21.15	21.21
52	21.44	21.49	21.55	21.60	21.66	21.72	21.78
53	22.02	22.06	22.12	22.19	22.24	22.30	22.36
54	22.62	22.68	22.74	22.80	22.86	22.92	22.98
55	23.22	23.28	23.34	23.40	23.46	23.52	23.58
56	23.84	23.90	23.96	24.03	24.09	24.15	24.21
57	24.48	24.53	24.59	24.66	24.72	24.79	24.85
58	25.12	25.18	25.25	25.32	25.38	25.45	25.51
59	25.78	25.85	25.91	25.99	26.06	26.12	26.19
60	26.46	26.53	26.60	26.67	26.74	26.81	26.88



Current Point	
DB	78
WB	58

Air Flow	1000
DB	78.000
WB	58.000
RH	28.1
W	40.1
v	13.674
h	25.001
DP	42.424
d	0.0731
vp	0.2722
AW	2.935

Off-line: STEP 3: Calculate

- Manual capacity calculation using total heat equation
 - $BTUh = 4.5 \times CFM \times \Delta h$ (field calculation)
 - $Tons = BTUh / 12,000$
- or with program
 - Program corrects for air density



Psychrometric Processes

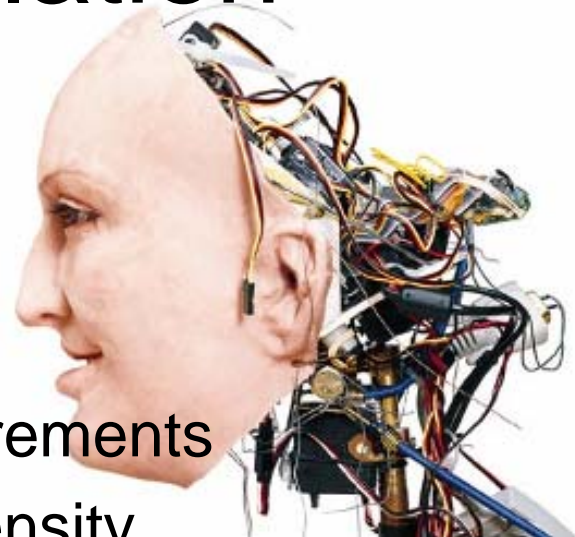
POINT	LABEL	AIR FLOW	UCM	PROCESS	POINT	DB	WB	RH	W	v	t	DP	h	VP	AW	START POINT	SECOND
entering	AR	1250	STD CFM	Add State Point	POINT	70.0	58.0	28.1	45.1280	13.67	25.88	42.4	0.8731	0.272	2.935		
exiting	AR	2500	STD CFM	Air Mixing	POINT	70.3	58.4	41.4	45.7695	13.58	24.81	45.8	0.8740	0.310	3.387	entering	entering

Start Point		Air Mixing (Second Point)		Current Point	
entering	entering	entering	entering	DB	70.331
Airflow (%)	100	Airflow (%)	100	WB	58.1
Air Flow	1250	Air Flow	1250	Air Flow	2500
DB	70.000	DB	70.900	DB	70.900
WB	58.000	WB	58.000	WB	57.993
RH	28.1	RH	28.1	RH	28.1
W	48.1	W	40.1	W	40.1
v	13.674	v	13.674	v	13.674
t	25.001	t	25.001	t	24.998
DP	42.424	DP	42.424	DP	42.405
h	0.8731	h	0.8731	h	0.8731
VP	0.2722	VP	0.2722	VP	0.2720
AW	2.935	AW	2.935	AW	2.932

Real-time calculation

Advantages:

- Very high accuracy
- Quick set up, real-time measurements
- Automatically adjusts for air density
- No charts or forms required
- All data used for calculation can be printed on site
- Trend and log data
- Download or Slave mode to PC via USB





Real-time system performance

- Process
 - The 435 calculates real time heating and cooling capacity
 - Using total heat equation
 - Compensates for air density changes
 - Humidity
 - Pressure (elevation)
 - Temperature
 - Can average, log, print or download the data
- Cost: ~\$1800





Integrated - Multifunction

- Equipment:
 - multifunction HVAC meter
 - Mini-vane anemometer probe in return
 - 2 wireless humidity probes
 - Before and after cooling coil/heat exchanger



*wireless
humidity
probe*



*Mini-vane
anemometer*



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*testo 435
multifunction
HVAC meter*



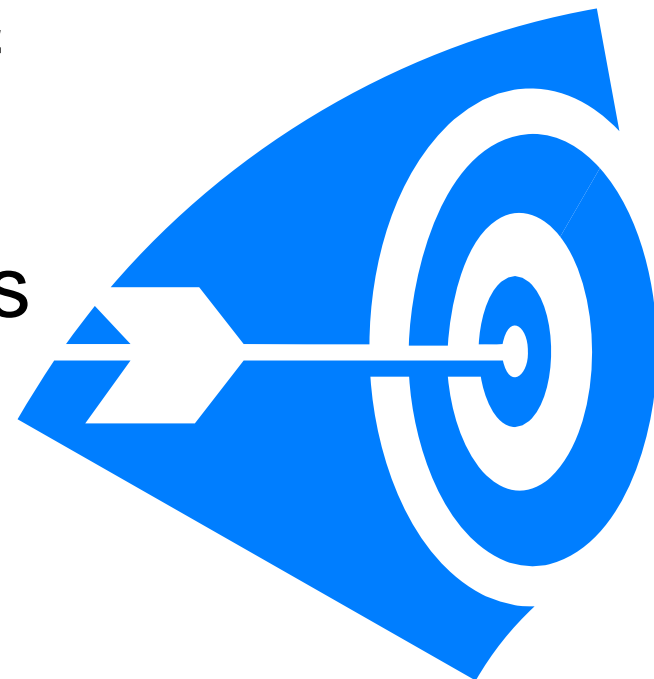


Accurate Measurements are Critical!

- Problems inherent with instrumentation lead to misdiagnoses.
- Technician after technician should get the same measured results.
 - Technicians should be able to make equipment operate in the field as well as it did in the lab!!! (You need “lab-accurate” instruments to do it!!!)

Impact of inaccuracy

- An error of ~ 1 °F wet bulb yields an error of 1/4 ton of cooling (3,000 BTU/hr)
- An error of 62.5 CFM yields an error of 1/4 ton of cooling (3,000 BTU/hr)



Thermometers and Psychrometers

- Thermometers
 - Sensor tolerances
 - Measurement technique
 - Sensor drift
- Sling Psychrometers
 - Proper air velocity
 - Distilled water
 - Low resolution
 - “Speed reading”
 - Sling in a duct?



Enabling technology

- Mini-Vane Airflow
 - Easy and fast
 - Accurate
 - No density compensation
 - Minimally invasive
- Humidity sensor
 - Accurate
 - Responsive
- Wireless probes
 - Real-time
 - Minimally invasive



Think clean

- Don't neglect the performance impact of dirt
Filters
 - A-coils
 - blower wheels
 - (secondary) heat exchangers
 - Burners
- And other unwanted stuff
- LBL report (2002)
 - Evaporator pressure drop doubles in 7.5 years
 - Siegel, Walker, Sherman





Contractor Study

- Wisconsin: 1 contractor co., 5 technicians
- Average summer temps like Philadelphia
- 85 tests: 37 TXV systems, 49 fixed orifice
- Capacity ranges

– 1.5 tons	6 systems
– 2.0 tons	52 systems
– 3.0 tons	23 systems
– 3.5 tons	1 system
– 4.0 tons	1 system
– 5.0 tons	2 systems

90% 2-3 tons



Energy Star on Air Flow



- Essential for comfort
 - 70% of systems tested are operating at less than 350 cfm/ton (ideal is 400 cfm/ton)
 - Annual savings of 8% possible
- Technician verifies system is flowing at 400cfm/ton (or cfm specified by manufacturer) during full-speed testing
 - Systems incapable of 350 cfm/ton or greater must be corrected by improving ducts or would not qualify

US Southwest/California study

Considering an ENERGY STAR
CAC/ASHP Specification
for 2006



Study results (CFM/ton)

	289-350	351-475	476-890
Pre test	5%	20%	76%
Post test	7%	65%	28%

Wisconsin study



Study results (CFM)

	3-9	10-99	100-299	300-499	500-985
decreased	3	2	17	4	2
increased	1				

57 systems left unchanged

Energy Star on proper charge.



Refrigerant Charge

- **Essential to maintain capacity**
 - Improper charge can lead to premature compressor failure
 - Up to 41% systems undercharged, 33% overcharged
 - Average savings of 12.5% with proper charge
- **Adjusted by technician in accordance with manufacturer's instructions**
 - Systems with more than $\pm 3^\circ$ deviation in subcooling from manufacturer's spec would not qualify

Considering an ENERGY STAR
CAC/ASHP Specification
for 2006



Study results (refrigerant charge)

	UNDER	GOOD	OVER
Pre-service	35%	35%	30%
Post-service	19%	53%	12%

65% wrong charge



Study results (Watts)

	0.1-9	10-99	100-299	300-499	500-699	700-916
saved		15	8	5	2	1
increased	3	4	2			

33% of systems saved from 10 to 500 watts
53% of systems left unchanged

Delivery

- Insure proper delivery at register
 - Volume flow & velocity (throw)
 - Large vane with CFM and averaging
 - Humidity
 - Accurate and responsive
 - Air Temperature
 - Large vane
 - With humidity



Assessment → Repair

- Heating & Cooling
 - Set air flow: using mini-vane
- Cooling
 - Correct refrigeration charge
- Heating
- Correct combustion set ups
 - Tuning
 - Fuel pressure
 - Venting
 - Safety





Resources

- Integrated Psychrometric program
 - <http://www.handsdownsoftware.com/downloads/testo-HDPsyChart-STUDENT.exe>
- testo A/C Applications Guide
 - Request PDF: bspohn@testo.com
- testo Combustion Applications Guide
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